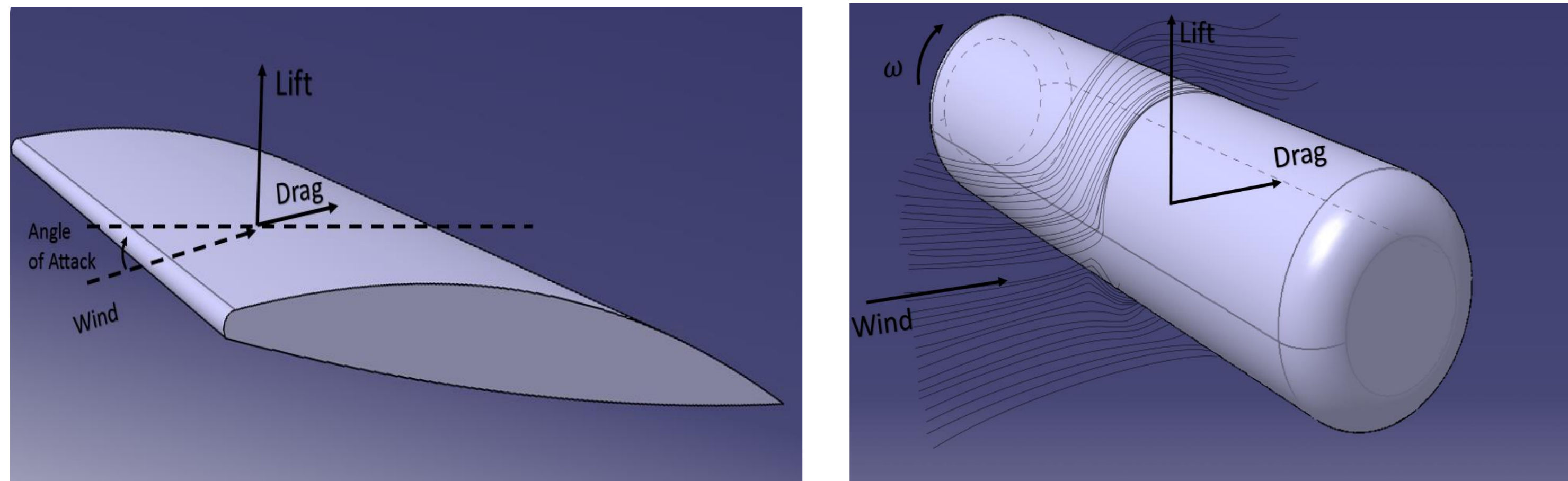


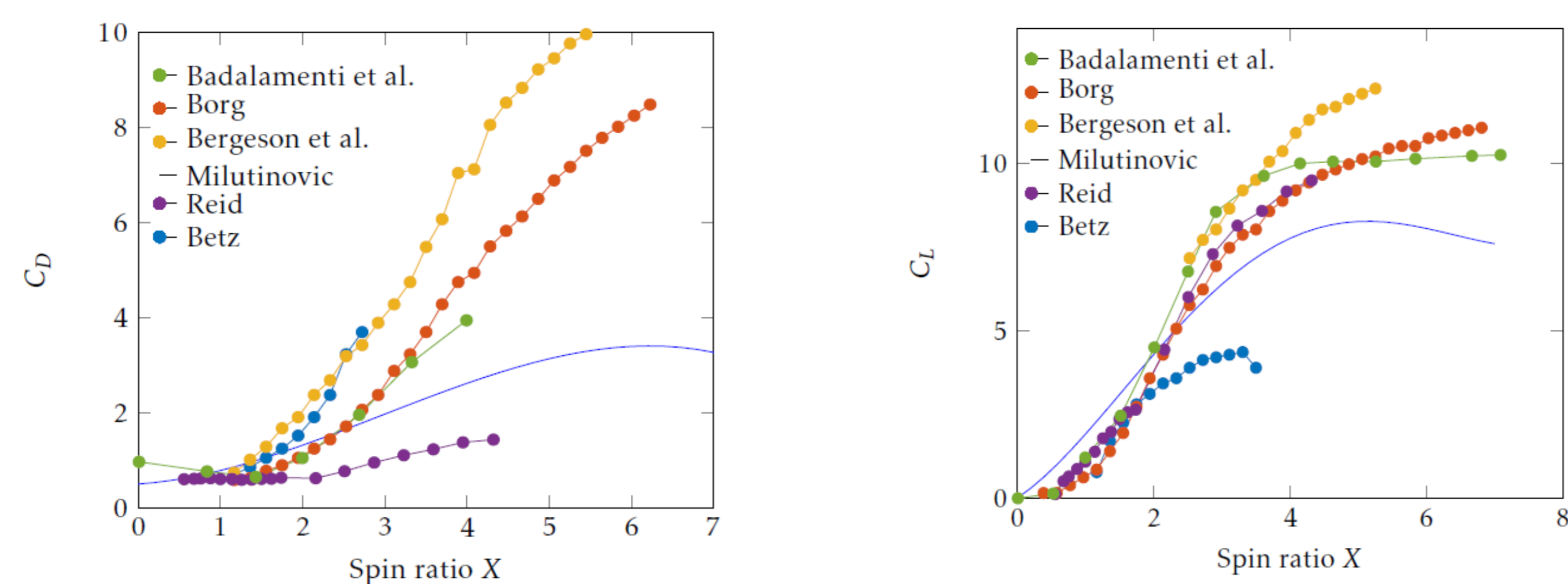
Introduction

Among the Airborne Wind Energy concepts Magnus based airborne wind energy systems uses rotating cylinders as aerostat. The rotating cylinder when exposed to wind flow produces a lift force, described as Magnus effect. The Magnus based aerostat have a high lift coefficient which is supplemented by lighter than air capabilities, and have a naturally robust design. The aerostat following a predefined trajectory leads to the development of high traction force in the tether which in turn is used to drive the generator and produce electricity.

Magnus Effect



C_L : Coeff. of Lift, C_D : Coeff. of Drag, X : Spin ratio $= \frac{\omega_{cyl} r_{cyl}}{v_{axz}}$, C_{Dy} : Coeff. of Drag- y_b - direction
 F_L : Lift Force $= \frac{1}{2} \rho S_{cyl} v_{axz}^2 C_L$, F_D : Drag Force $= \frac{1}{2} \rho S_{cyl} v_{axz}^2 C_D$,
 F_{Dy} : Drag Force, y_b - direction $= \frac{1}{2} \rho S_{cyl} v_{ay}^2 C_{Dy}$

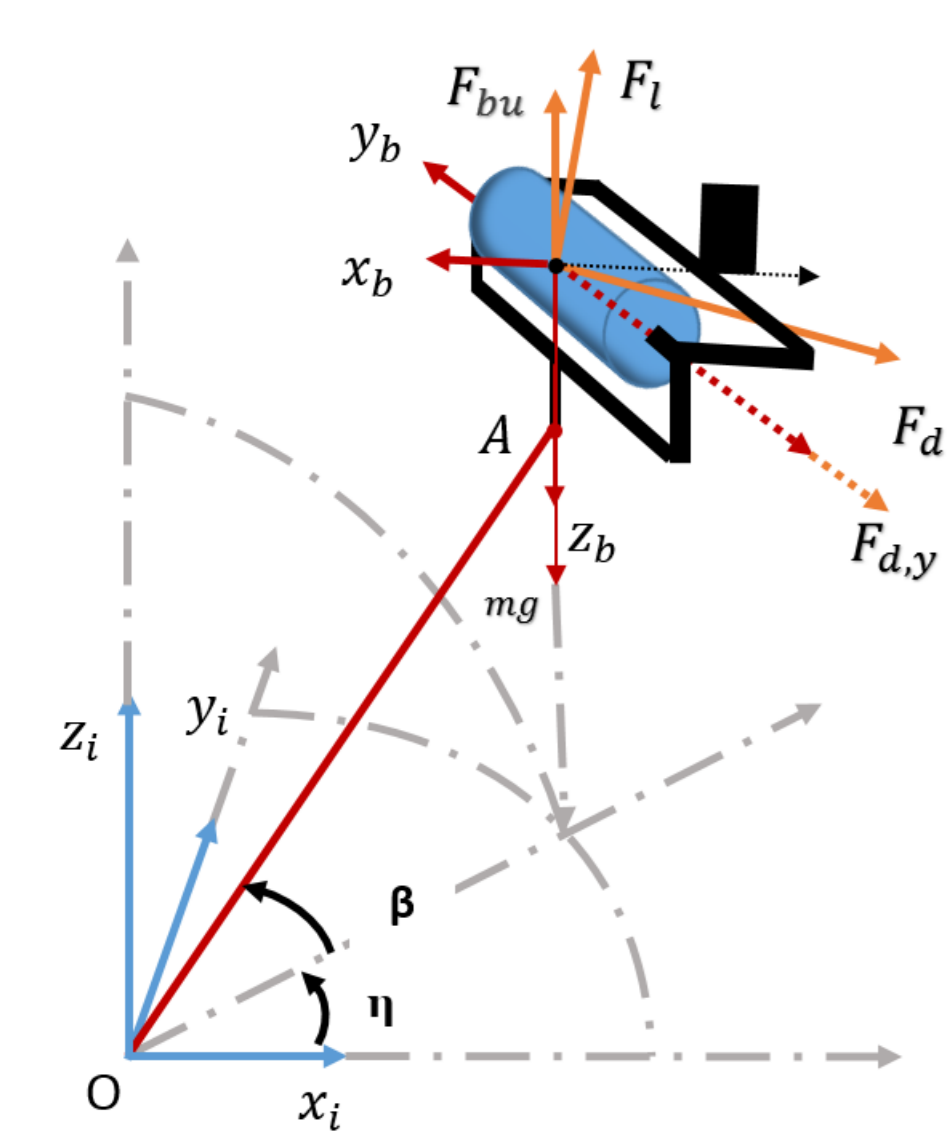


The above analysis shows that the assumed polynomial expressions for the Coeff. of Lift (C_L) and the Coeff. of Drag (C_D) i.e. the aerodynamic model for Magnus cylinder as proposed by Milutinovic [1] agrees with the historical experimental data available on Magnus cylinder.

$$C_D = -0.0211X^3 + 0.1873X^2 + 0.1183X + 0.5,$$

$$C_L = 0.0126X^4 - 0.2004X^3 + 0.7482X^2 + 1.3447X$$

Mathematical Model



Equation of rate of change of translational velocity [2]

$$\dot{\mathbf{v}}_b = \frac{1}{m} (\mathbf{F}_b - \tilde{\omega}_b \mathbf{v}_b)$$

where,

$$\tilde{\omega}_b = \begin{bmatrix} 1 & -r & q \\ r & 0 & -p \\ -q & p & 0 \end{bmatrix}, \text{ and}$$

\mathbf{F}_b represents Body forces acting on the ABM and is given by

$$\mathbf{F}_b = \mathbf{F}_L + \mathbf{F}_D + \mathbf{F}_{dy} + \mathbf{W}_b + \mathbf{F}_{bu} + \mathbf{F}_r$$

\mathbf{W}_b : Weight in Body Frame, \mathbf{F}_{bu} : Bouyant Force, \mathbf{F}_r : Rope Force, $\{x_b - y_b - z_b\}$: Body frame of ref., $\{x_i - y_i - z_i\}$: Inertial frame of ref.
 p : Roll rate, q : Pitch Rate, r : Yaw Rate, β : Elevation angle, η : Azimuthal angle

Static Model

- Theoretical Power produced during production phase (P_{prod}) as proposed by [3] Loyd (1980) and refined in [4] Argatov et al. (2009)

$$P_{prod} = \frac{1}{2} \rho 4 S_{cyl} \left(\frac{v_{\omega} \cos(\beta)}{3} \right)^3 C_L \left(\frac{C_L}{C_D} \right)^2, \quad \dot{r}_{prod} = \frac{v_{\omega} \cos(\beta)}{3}: \text{Reel-out speed}$$

- Theoretical Power consumed during recovery phase (P_{rec})

$$P_{rec} = \frac{1}{2} \rho S_{cyl} (v_{\omega} \cos(\beta) + \dot{r}_{rec})^2 C_{Drec} \dot{r}_{rec}, \quad \dot{r}_{rec}: \text{Reel-in speed}$$

- Estimated Power produced in one complete cycle (P_{cycle})

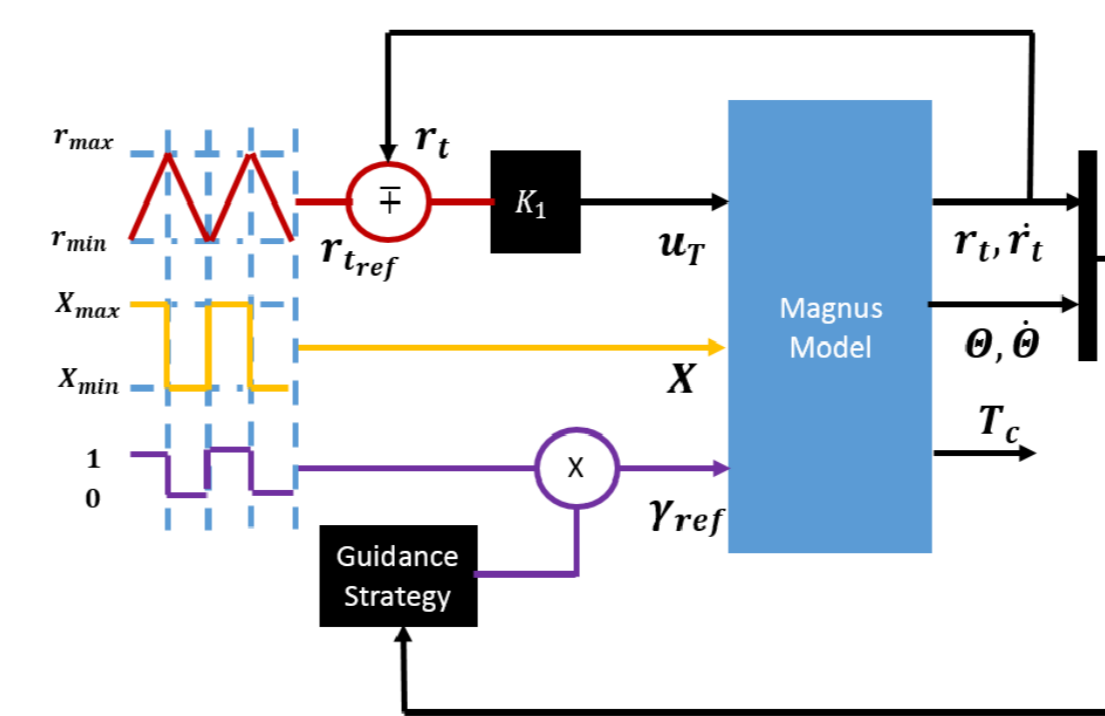
$$P_{cycle} = \frac{P_{prod} \dot{r}_{rec} - P_{rec} \dot{r}_{prod}}{\dot{r}_{rec} + \dot{r}_{prod}}$$

- Hence, to maximize the power is to maximize the ratio $C_L \left(\frac{C_L}{C_D} \right)^2$, the maximum $C_L \left(\frac{C_L}{C_D} \right)^2$ for the magnus cylinder is found to be at spin ratio, $X = 3.6$.

Control Strategy

Guidance strategy

- We apply the guidance strategy given in [5], and another gain k_{η} to obtain a constant width trajectory $\eta_{ref} = \frac{k_{\eta}}{r_t}$.

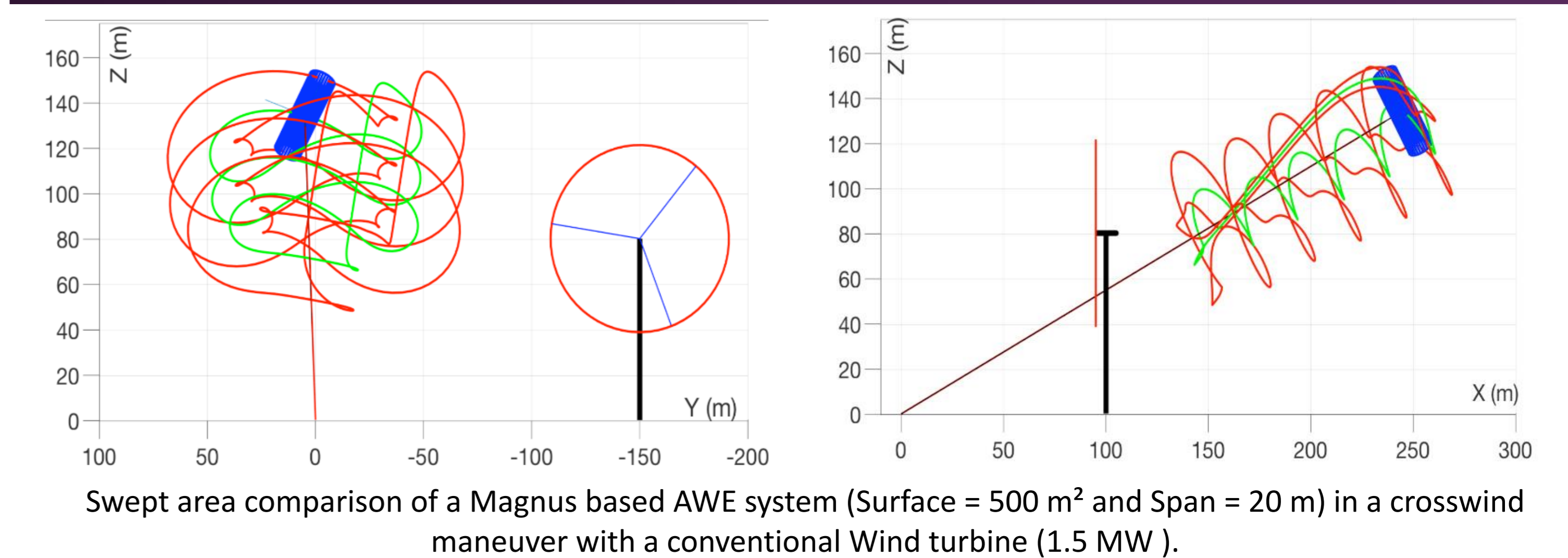


Control of tether length

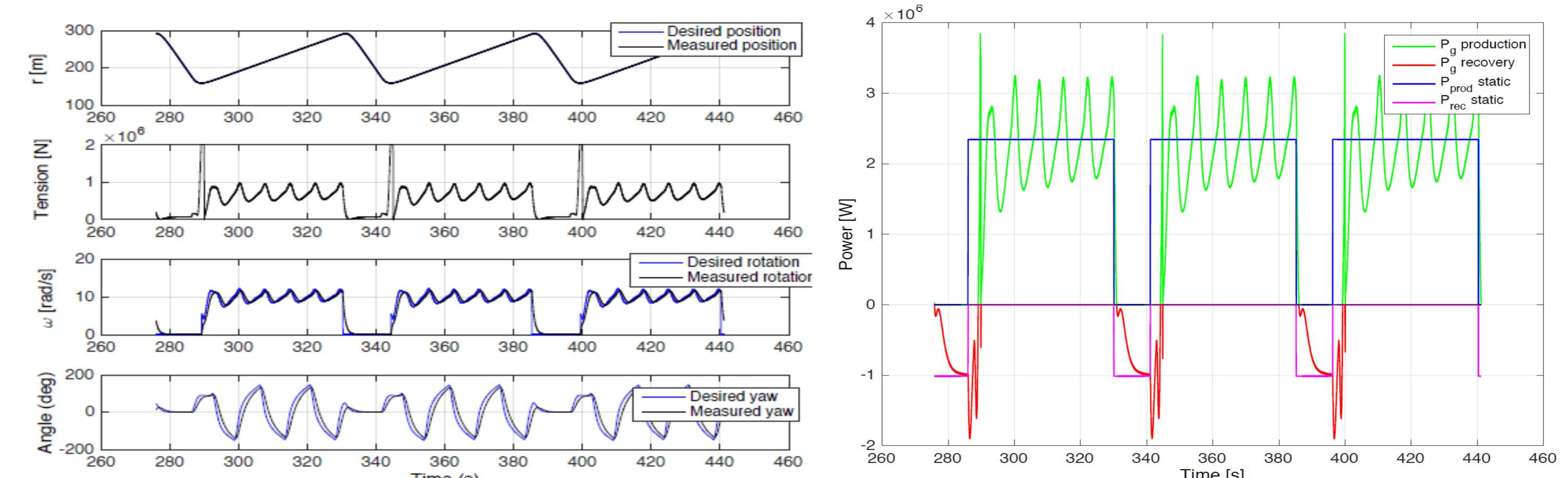
- A PID controller K_1 is used in order to follow the radial position r_{tref} through U_T acting on the winch actuator.
- The response time for this control loop is set to be faster than the variations of other forces in order to get an efficient production cycle.

Θ : Attitude of Magnus cylinder by ZYZ $\{\alpha, \delta, \gamma\}$
 T_c : Winch Tension
 r_t : Tether length
 γ_{ref} : Yaw angle in ZYZ transformation
 r_{tref} : Reference radial position

Simulation Results



Swept area comparison of a Magnus based AWE system (Surface = 500 m² and Span = 20 m) in a crosswind maneuver with a conventional Wind turbine (1.5 MW).

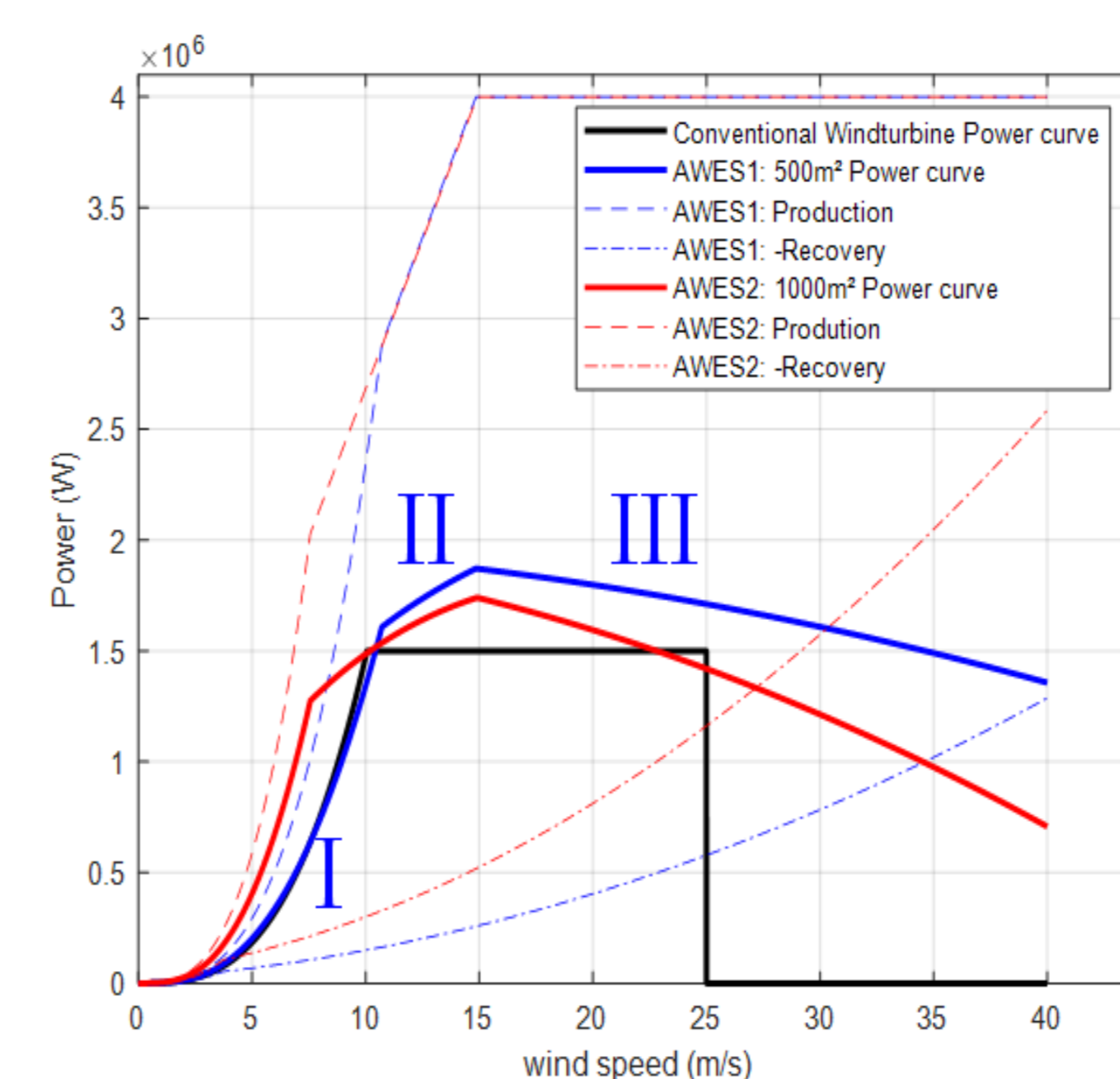


Reference and state variable for tether length (r_t , r_{tref}), tether tension T_c , angular speed of the Magnus rotor (ω_{cyl} , ω_{cylref}), and yaw angle (γ , γ_{ref}).

Simulated output power during production and recovery phases with a comparison with a simplified static model P_{static} .

Power Curves

Comparison of Power Curve based on static model of a Magnus-based AWE system with that of a conventional Wind turbine (1.5MW).



Phase I: Power extraction is maximized following Loyd cond.
Phase II: Maximum traction force is reached, \dot{r}_{prod} continues to increase.
Phase III: Maximum speed of the generator is reached.

By modifying Surface (S_{cyl}), Maximum Tension (T_{max}), and Maximum Power (P_{max}), the shape of the power curve can be adapted according to the distribution of the wind speed at the site.

Conventional Wind turbine : 1.5MW
 AWES1: $S_{cyl} = 500 \text{ m}^2, T_{max} = 8e5 \text{ N}, P_{max} = 4 \text{ MW}$
 AWES2: $S_{cyl} = 1000 \text{ m}^2, T_{max} = 8e5 \text{ N}, P_{max} = 4 \text{ MW}$

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